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## Tip Loss Factor Effects on Aerodynamic Performances of ASSESSMENT THE FEASIBILITY OF USING THE HEAT DEMAND-OUTDOOR Tip Loss Factor Effects on Free ay namic Performances of Horizontal Axis Wind Turbine Horizontal Axis Wind Turbine

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## **Abstract**

for which the maximum power coefficient is calculated at different design tip speed and glide ratio. Our simulation is conducted  $\mathcal{L}$  and  $\mathcal{L}$  addressed in the most effective solutions for decreasing the most e for maximum power production. This paper presents the optimal tip design using the new tip loss correction. A semi-analytical solution was proposed to find the optimum rotor considering Shen's new tip loss model. The optimal blade geometry is obtained for S809 rotor wind turbine blade type, produced by National Renewable Energy Laboratory (NREL). Tip loss corrections are an important factor in Blade Element Momentum (BEM) theory when determining optimum blade design

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Converte: BEM mathod, S200 sigfoil. Herizontal axis wind turbing. Aerodynamic performances. Tip speed ratio, nower coefficient Keywords: BEM method, S809 airfoil, Horizontal axis wind turbine, Aerodynamic performances, Tip speed ratio, power coefficient

#### renovation scenarios were developed (shallow, intermediate, deep). To estimate the error, obtained heat demand values were **1. Introduction**   $T_{\text{total}}$  of wind turbines rotor was made by contractional codes must be able to give  $\mu$

The design of wind turbines rotor was made by computational codes must be able to gives the best wind turbine geometry in order to obtain the maximum power. Computational fluid dynamic code (CFD) is widely used, to produce the accurate results, but on the other need longer time for calculation and big informatics memory. The mathematical model most frequently used by industrial and scientific research, this code based on Blade Element Momentum Theory (BEMT), it gives less accurate results, but it is possible to determine the optimal rotor geometry with maximum power design and evaluate the forces and torque acting on the blades. The optimal design of blade couplet suggested on agreed personators. Whilst Navior Stelse solvers can be incorporated into roter optimization geometry is based on several parameters. Whilst Navier-Stokes solvers can be incorporated into rotor optimization

buildings that vary in both construction period and typology. The scenarios (low, medium, high) and three distriction  $\mathcal{L}_\mathbf{r}$ 

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for greater accuracy, BEM computations have significant advantages in computational speed and ease of implementation. An alternative to using a more computationally intensive method is to modify the BEM model and apply corrections to the technique. One of the most important corrections to BEM analysis is a tip loss correction. The concept of a tip loss was introduced by Prandtl [1] to simplify the wake of the turbine by modeling the helical vortex wake pattern as vortex sheets that are convected by the mean flow and have no direct effect on the wake itself. This theory is induced velocity field, F. This correction is used to modify the momentum of the blade element momentum equations. One lilitation of this model is that it assumes the wake does not expand, limiting its validity to lightly loaded rotors.

Another limitation of the BEM theory is that when the axial induction factor is greater than 0.4, the basic theory becomes invalid. This occurs with turbines operating at high tip speed ratio, for this Glauert [2] developed a correction to the trust coefficient based on experimental measurements with large induced velocities. While this model was originally developed as a correction to the thrust coefficient of a rotor, it has also been used to correct the local coefficient of the individual blade elements when used with BEM theory. Because of this, it is important to understand the Glauert correction's relationship to the tip-loss model. When the losses near the tip are high, the induced velocities are large; therefore, the possibility of a turbulent wake near the tips increases. Thus, for each element the total induced velocity calculation must use a combination of the tip-loss and Glauert corrections. A new modification derived by [3] to the Glauert empirical relation that included a new formulation of the tip-loss correction. Shen et al. [4] corrected both the induced velocities and the mass flux for tip loss effects, and corrected the lift coefficient by introducing a new factor. Others corrections introduced by [5-7] for BEMT solver for two such phenomena: tip-hub loss factor and high induction factor conditions. The multi-objective and Computational Fluid Dynamics (CFD) methods [8, 9] have been used for maximizing the lift-to-drag ratio, minimizing the sound pressure and flox analysis using XFoil and NAFNoise programs.

The objective of the current work is to maximise the power extraction efficiency of a wind turbine rotor, with a focus on the effect of various tip loss models on the optimal rotor shape and introduce a new formulation of axial and angular induction factors.

# **2. Mathematical model**

As the classical theory of wind turbine rotor aerodynamic, the BEM method combines the momentum and blade element theory. The blade is divided into several elements, by applying the equations of momentum and angular momentum conservation, for each element dr section of the blade, axial force and torque can be defined by Eqs. (1) and (2), respectively:

$$
dT = \frac{1}{2} B \rho c V_{\text{rel}}^2 C_n dr \tag{1}
$$

$$
dM = \frac{1}{2} B\rho c V_{\text{rel}}^2 C_t r dr \tag{2}
$$

These relations from the momentum theory alone do not include the effects of blade shape, for it, the blade element theory was introduced. Accordingly, the angle of relative wind,  $\phi$  and twist angle is determined by Eqs. (3) and (4) respectively:

$$
\tan(\phi) = \frac{(1-a)V_0}{(1+b)\Omega r}
$$
\n(3)

$$
\beta = \phi - \alpha_{op} \tag{4}
$$

Where  $\alpha_{\rm op}$ , is the optimal angle of attack, it extracted from 2D CFD calculation.

The turbine has B number of blades. Therefore, the force of thrust and torque at each element *dr* given by Eq. (5) and (6) respectively:

$$
dT = 4\pi B \rho V_0^2 a (1 - a) r dr \tag{5}
$$

$$
dM = 4\pi B\rho V_0 \Omega b (1-a)r^3 dr \tag{6}
$$

By equating (1) with (5) and (2) with (6), the axial and tangential induction factors can be found as follows:

# ِ متن کامل مقا<mark>ل</mark>ه

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