ARTICLE IN PRESS

Renewable and Sustainable Energy Reviews xxx (xxxx) xxx-xxx

ELSEVIER

Contents lists available at ScienceDirect

Renewable and Sustainable Energy Reviews

journal homepage: www.elsevier.com/locate/rser



Solar heating and cooling systems by absorption and adsorption chillers driven by stationary and concentrating photovoltaic/thermal solar collectors: Modelling and simulation

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ARTICLE INFO

Keywords:

Solar heating and cooling systems Dynamic energy performance analysis Concentrating photovoltaic/thermal panels Flat plate photovoltaic/thermal panels Adsorption and absorption chillers

ABSTRACT

Solar heating and cooling systems are a promising technology which may significantly contribute to the reduction of greenhouse gas emissions, the enhancement of energy efficiency, and the increase of renewables share in the building sector. The available literature show a high number of papers aiming at investigating solar heating and cooling systems based on heat driven and solar technologies, configurations, operating strategies, and financing issues. Nevertheless, none of the papers available in literature investigates the possibility to replace conventional solar thermal collectors by flat plat and concentrating photovoltaic/thermal systems, also producing renewable electricity. To cover this lack of knowledge, in this paper a dynamic simulation model of novel solar polygeneration heating and cooling systems is presented. Such dynamic simulation model is developed and implemented in a computer code, written in MatLab, and allows investigating the energy, economic and environmental performance of such novel solar polygeneration systems, based on both adsorption and absorption chiller technologies fed by dish-shaped concentrating and flat photovoltaic/thermal collectors. In order to show the potentiality of the presented tool, a comprehensive parametric case study is carried out to find out the optimal system configurations, as a function of crucial design and operating parameters and of weather conditions. The presented case study analysis refers to a small cluster of four buildings, including office and residential spaces, located in different European weather zones. The modelled solar polygeneration systems simultaneously produce electricity, space heating and cooling, and domestic hot water; electricity is self-consumed or delivered to the electrical grid. For comparative purposes, two different back-up system configurations, based on an electric chiller and a condensing gas-fired heater are also taken into account as conventional reference building-plant systems.

By means of this systematic parametric analysis, comprehensive guidelines for system designers, practitioners and/or researchers focused on the development and use of solar heating and cooling systems are provided.

1. Introduction

1.1. Literature review

In the last decades, a significant effort has been paid towards the sustainable development, with a particular attention to environmental issues, especially to those associated with energy use and production. In the field of sustainable development, the building sector plays a crucial role, showing a high potential of energy savings [1–6]. To reduce the primary energy consumption due to space heating and cooling in residential and commercial buildings, more effective policies towards the challenging goals of greenhouse gas emissions reduction,

energy efficiency improvement, and increase of renewables share, have been defined, such as those arising from the recent global agreements among countries on new national emissions targets (e.g. COP21) [7–9].

In order to reach these goals, researchers and institutions, especially in Europe, are devoting a particular effort in the investigation of effective measures for the energy efficiency and the development of renewable energy based technologies for building applications, necessary to provide the highest environmental benefits [10–13]. Among the available technologies, supported by global actions to improve renewable energy access, Solar Heating and Cooling (SHC) systems have attained a significant attention. Such technology is capable to greatly exploit the solar energy to provide space heating and cooling all over

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https://doi.org/10.1016/j.rser.2017.10.059

 $1364\text{-}0321/\ @$ 2017 Published by Elsevier Ltd.

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Nomenclature		U	heat transfer coefficient (W/m ² K)
Α	Solar collector gross area (m ²)	v V	ratio of TK1 volume to solar field surface area (l/m²) Volume (m³)
a_1	global heat loss coefficient [W/(m ² K)]	v WCH	Water-cooled electric Chiller
a_1	temperature dependence of the global heat loss coefficient	WGH	water-cooled electric chinici
u ₂	$[W/(m^2 K^2)]$	Subscripts	
AbCH	bsorption CHiller		
AdCH	Adsorption Chiller	a	outdoor environment
AH	Auxiliary gas-fired Heater	abs	absorber plate
C	geometric concentration ratio (-)	act	activation
C^{TK}	Tank thermal capacitance (kJ/K)	AbCH	Absorption CHiller
_	specific heat capacity (kJ/kg K)	AdCH	Adsorption Chiller
c _p	unitary cost (€/kWh)	aux	auxiliary
CDD	Cooling Degree Days (Kd)	b	beam
CI	Investment capital cost (€)	chill	chilled
COP	Coefficient of Performance (-)	cool	cooling
CPVT	Concentrating PhotoVoltaic/Thermal collectors	CP	Compressor
CT	Cooling Tower	el	electrical
DNI	Direct Normal Irradiation (kWh/m²)	Ev	Evaporator
DHW	Domestic Hot Water	fit	feed in tariff
E	Electricity (W)		gas
h	specific heat transfer coefficient (W/m ² K)	g gl	glazed PVT collector
f	ratio of absorption chiller capacity to peak cooling load (-)	gle	glazed PVT collector with low-e coating
F'	collector efficiency factor (-)	gro	gross
FGE	Electricity fed to the grid (kWh)	i	inlet
G	incident solar radiation (W/m ²)	inv	inverter
GTI	Global Tilted Irradiation (kWh/m²)	HT	High Temperature
HDD	Heating Degree Days (Kd)	heat	heat
ISR	incident global horizontal Solar radiation (kWh/m ² y)	load	load
LHV		loss	thermal losses
цп v m	Low Heating Value (kWh/Nm³)	L	Power
	fluid mass flow rate (kg/s)	L LT	Low Temperature
$MEFG_{PV}$	economic profit related to the electricity fed to the grid	m	average
MECC	(€/y)	max	maximum
$MESC_{PV}$	economic saving related to the buildings self-consumed	n	optical
NOC not	electricity (\mathfrak{C}/y) operating cost of the proposed system layout (\mathfrak{C}/y)	nel	conventional average national electricity
NPEC	Net Primary Energy Consumption (kWh)	0	outlet
OC			
	Operating Costs (€/y)	opt	optic
P	Power (W)	par	parasitic
PEC	Primary Energy Consumption (kWh)	PV	photovoltaic
PES	Primary Energy Consumption (kWh)	ref	reference
PS1	Proposed System including an electric chiller as auxiliary	s SC	summer Solar collector
DCO	device		
PS2	Proposed System including a gas fired heater as auxiliary	SP	Set Point
DI /m	device	STC	standard test conditions
PVT	PhotoVoltaic/Thermal collectors	th	thermal
q	thermal flux (W/m²)	TOT	total
Q	thermal power (W)	ugl	unglazed PVT collector
RS	Reference System	useful	useful energy
S	ratio of solar field surface area to the peak cooling load (m^2/kWf)	W	winter
SC	Solar collector field area (m ²)	Greek sy	mbols
SCE	Self-consumed electricity (kWh)		
SF	Solar Fraction (-)	α	ratio of solar field surface area to chiller capacity (m ² /kW
SHC	Solar Heating and Cooling	β	the temperature coefficient on power (K ⁻¹)
SPB	Simple PayBack (y)	η	efficiency (-)
T	temperature (K)	τα	effective transmittance-absorbance product (-)
TK	Storage tank		

the year, significantly contributing to achieve the above mentioned challenging targets [14]. Particularly during summer, solar cooling systems are able to time shift the space cooling energy demand from electricity to solar energy, which is efficiently and simultaneously exploitable with low or negligible environmental impact [15]. In SHC

systems, solar thermal energy is generally obtained by means of solar thermal collectors (flat plate, evacuated tubes, parabolic trough, etc.) [16–18]. Produced thermal energy may be directly used for heating purposes and domestic hot water preparation or supplied to thermally-driven chillers for cooling energy production [14,19]. As a result, SHC

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