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Position based simulation of solids with accurate contact handling

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ABSTRACT

Simulating multi-body dynamics with both rigid and flexible parts and with frictional contacts is a hard problem. We solve this by expressing the couplings between the bodies as position level constraints. The implicit treatment of the constraint directions gives us improved stability over velocity based methods. Then by employing regularization of nonlinear constraints and a convex minimization formulation, we bridge constraint-based methods to traditional force-based methods. In fact, the former are just a dual variables formulation of the latter. We solve this dual problem using position based dynamics (PBD). We show how PBD is a completely valid modeling technique and we extend it with an accurate contact and Coulomb friction model. We further show for the first time how the same solver can be used to simulate both rigid and deformable solids with two way coupling. For the soft bodies we introduce a novel form of linear finite elements expressed as constraints, that is more accurate than PBD mass-spring systems. More of our results include the energy conserving Newmark integrator and the accelerated Jacobi solver suitable for parallel architectures. Note that this paper is an extended and revised version of the conference paper published in [1].

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1. Introduction

For the last decade position based dynamics (PBD) has been successfully applied to the simulation of particle systems and deformable bodies [2]. This was possible given the inherent stability of the method due to its full implicit formulation: not only the magnitude of the constraint forces are considered implicit, but also their directions [3]. This is especially true for materials with fast changing constraint gradients and transverse oscillations, e.g. cloth or threads [4].

At its heart, PBD is a nonlinear constraint projection scheme, similar to the ones used in molecular dynamics [5]. The main drawback of PBD is that it has no rigorous mathematical model for contact and friction and thus it is almost never used for rigid body simulations (with the exception of [6]). In our literature research we have not found any clear proof for the convergence of a PBD-like method with unilateral constraints and friction. Because of this, some authors choose to treat contacts as bilateral constraints [3] or approximate friction at the end of the step [7,8] without giving a sound recipe for mixing friction with the position corrections. This paper reiterates our existing work in [1] and brings some extensions to it.

We offer not only an accurate treatment of contact and friction, but we also simulate deformable bodies in a physically cor-

rect manner using the theory of continua and the finite element method (FEM). By employing the constraint regularization technique [9] at position level, we are able to show that our constraint solving problem is just the dual variables formulation of an equivalent elasticity problem. In the end, we are able to unify the simulation of rigid and deformable solids under the umbrella of PBD, using constraints as building blocks.

1.1. Related work

There has been a wealth of work published on the subject of rigid body simulation with contact and friction - for a survey see [10]. We note the advances made in the 90s by Baraff, Stewart and Anitescu. Given the drawbacks of penalty forces, Baraff introduced the acceleration based linear complementarity problem (LCP) method. This method had its problems too (related to impacts and the Painlevé paradox) that were later solved by a velocity based approach that allows discontinuities in the velocities, i.e. impulses. The new velocity time stepping (VTS) schemes [11,12] became very popular in computer graphics, games and real time simulators. We take a similar approach in this paper, but based on more recent work geared towards convex optimization [13–15], although expressing the problem as such an optimization is not mandatory.

Traditionally in computer graphics deformable bodies have been simulated using implicit integrators due to their unconditional stability properties. These have been applied not only to

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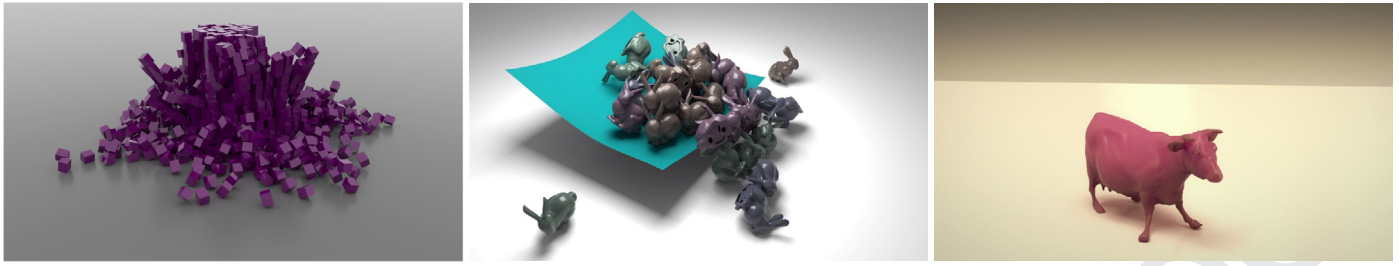


Fig. 1. From left to right: rigid boxes falling on ground, bunnies falling on a piece of cloth, a flexible cow falling on ground.

49 mass-spring systems, but also to simulations using finite differ- 100
50 ences or the finite element method (FEM) [16]. Recently, the pop- 101
51 ular Backward Euler integrator has been recast as an optimization 102
52 problem [17] helping us to gain new insights. 103

53 While initially constraint based methods were not considered 104
54 for simulating deformable bodies, this changed with the advent of 105
55 PBD [7] and constraint regularization [9]. PBD was originally intro-
56 duced by Jakobsen for games based on molecular dynamics meth-
57 ods and a nonlinear version of the Stewart–Trinkle solver for rigid
58 bodies [8]. Goldenthal later showed how PBD stems from the fully
59 implicit integration of a constrained system [3]. Even though in
60 theory constraints do not allow deformation for all the degrees of
61 freedom (often resulting in *locking* [18]), in practice, they proved
62 quite successful for simulating a wide range of objects (e.g. cloth,
63 hair, soft bodies - see [2] for a survey). This is due to the fact that
64 iterative solvers are often not run to convergence and this makes
65 the constraints soft.

66 The idea of a unified solver is not new and our simulator bears 106
67 maybe most similarity to Autodesk Maya’s Nucleus. Our results are 107
68 also along the line of more recent PBD work [6,19–21] and Projec-
69 tive Dynamics [17].

70 A great job of emphasizing the role of nonlinearity for achiev- 108
71 ing stability was done in [22] - or rather the importance of using 109
72 a full implicit integration of nonlinear forces. Keeping the implicit 110
73 formulation intact is also the idea in [4] and the fact that dissip- 111
74 ation (even if artificial) is key to the stability of the system is 112
75 stressed in [23]. We pursue a similar approach, but rely mostly on 113
76 updating the constraint directions at every iteration, without alter- 114
77 ing the mass matrix. 115

78 1.2. Contributions 116

79 We aim in this paper to show that PBD is a physically sound 117
80 method. This is done in Section 2 where we introduce a fixed point 118
81 iteration and prove it converges in Appendix A. Also, PBD can be 119
82 used for both rigid and deformable bodies with constraints, contact 120
83 and friction in a single unified solver. The advantages of this for- 121
84 mulation include better constraint satisfaction, improved stability 122
85 and out of the box two way coupling of rigid and elastic materials. 123
86 In addition to [1], we explain more in depth why the method is so 124
87 robust and how it can be made more conservative while maintain- 125
88 ing its stability properties. Another new sub-section explains how 126
89 our method is not bound to a minimization formulation and can 127
90 also be recast as a fixed point iteration of a box LCP solver. 128

91 Our new viscoelastic model permits us to incorporate soft 129
92 constraints, damping and FEM into PBD - for applications see 130
93 Section 3. Another goal we had in mind was to keep the computa- 131
94 tional overhead to a minimum compared to existing methods. This 132
95 is why we chose our mathematical formulation to be expressible 133
96 as a matrix-free solver. We present a novel projected gradient de- 134
97 scent algorithm for nonlinear optimization in Section 4. The algo- 135
98 rithm is based on both the Jacobi and the Nesterov methods so 136
99 it can be parallelized. In Section 5 we continue to give some more 137

100 details on how to implement this solver (or a Gauss–Seidel one) for 101
101 specific examples like the frictional contact constraint or the FEM 102
102 tetrahedron constraint. In the end we give some code implementa- 103
103 tion notes and take a closer look at our results. As an extension to 104
104 [1], we added some extra figures, plots and comments proving the 105
105 nice stability and energy conservation properties of our solver.

106 2. Mathematical model 107

108 2.1. Equations of motion 109

109 In this section we present the continuous equations of motion 110
110 and a way to discretize them that will be the basis of our further 111
111 developments. We start with the equations of motion for a gener- 112
112 al system of bodies and, at first, we also introduce bilateral con- 113
113 straints between the bodies: general nonlinear functions equated 114
114 to zero, describing for example a bead on a wire or joints articulat- 115
115 ing rigid bodies. The resulting equations can also be derived from 116
116 Hamilton’s principle and the principle of virtual work by using a 117
117 Lagrangian augmented by a special constraint potential: $-\gamma^T \Psi(\mathbf{q})$ 118
118 (DAE) [24]

$$119 \mathbf{M}\dot{\mathbf{v}} = \mathbf{f}_{tot} + \nabla \Psi(\mathbf{q})\boldsymbol{\gamma}, \quad (1)$$

$$120 \dot{\mathbf{q}} = \boldsymbol{\zeta}(\mathbf{q})\mathbf{v}, \quad (2)$$

$$121 \Psi(\mathbf{q}) = \mathbf{0}, \quad (3)$$

122 where $\mathbf{v} \in \mathbb{R}^n$ is the generalized velocity vector, n is the number of 123
123 degrees of freedom of the system, $\mathbf{q} \in \mathbb{R}^{n'}$ is the generalized posi- 124
124 tion vector ($n' \geq n$ is the optimal number of parameters describing 125
125 position and orientation), $\boldsymbol{\zeta}$ is a linear kinematic mapping between 126
126 velocities and position derivatives, \mathbf{M} is the mass matrix [10], $\Psi(\mathbf{q})$ 127
127 is a vector-valued bilateral constraint function, $\nabla \Psi(\mathbf{q})$ is its gradi- 128
128 ent (i.e. the constraint directions), $\boldsymbol{\gamma} \in \mathbb{R}^m$ is a Lagrange multipliers 129
129 vector enforcing the bilateral constraints in (3) (m is the number 130
130 of constraints), and \mathbf{f}_{tot} is the total generalized force acting on the 131
131 system (external and Coriolis). 132

132 In order to discretize the equations of motion we use the Im- 133
133 plicit Euler (IE) integrator

$$134 \mathbf{M}(\mathbf{v}^{l+1} - \mathbf{v}^l) = h \nabla \Psi(\mathbf{q}^{l+1})\boldsymbol{\gamma}^{l+1} + h\mathbf{f}_{tot}^l, \quad (4)$$

$$135 \mathbf{q}^{l+1} = \mathbf{q}^l + h\mathbf{L}\mathbf{v}^{l+1}, \quad (5)$$

$$136 \Psi(\mathbf{q}^{l+1}) = \mathbf{0}, \quad (6)$$

137 where l is the current simulation frame, h is the time step (consid- 138
138 ered constant), and $\mathbf{L}(\mathbf{q}^l)$ is a linear kinematic mapping [10] with 139
139 $\mathbf{L}^T \mathbf{L} = \mathbf{1}$ (the identity matrix). The IE discretized equations can be 140
140 brought to a minimization form

$$141 \mathbf{v}^{l+1} = \arg \min_{\Psi(\mathbf{q}^l + h\mathbf{L}\mathbf{v}) = \mathbf{0}} \frac{1}{2} \mathbf{v}^T \mathbf{M} \mathbf{v} - \hat{\mathbf{f}}^T \mathbf{v}, \quad (7)$$

142 where $\hat{\mathbf{f}} = \mathbf{M}\mathbf{v}^l + h\mathbf{f}_{tot}^l$ and the new positions come from (5). 143

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