

Contents lists available at ScienceDirect

Mechanism and Machine Theory

journal homepage: www.elsevier.com/locate/mechmachtheory



Research paper

Formulation of five degrees of freedom ball bearing model accounting for the nonlinear stiffness and damping of elastohydrodynamic point contacts



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ARTICLE INFO

Article history: Received 25 January 2018 Revised 14 February 2018 Accepted 2 March 2018

Keywords:
Ball bearings
Nonlinear contact model
Angular contact
EHD lubrication

ABSTRACT

The knowledge of the dynamic behavior of rolling elements bearings and the contact properties have shown to be essential for model based fault diagnosis, lifetime estimation and reduction of noise in several industrial equipment application. This paper formulates the force and moments equilibrium of an angular contact ball bearing with five degrees of freedom accounting for the effects of the elastohydrodynamic (EHD) lubrication of its elements. A complete nonlinear model is derived and equivalent parameters for stiffness and damping of each contact were evaluated for different loading conditions. An iterative solution process was proposed to couple the bearing equilibrium and the EHD contact calculation, so as to generate the most suitable representation of the lubrication condition at the equilibrium point. The resulting reduced order model is promising to save computational costs and to make feasible the analysis of complex rotating systems supported by oil lubricated angular contact ball bearings.

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1. Introduction

Due to the increasing interest on model-based fault diagnostics, overall precision improvement and noise reduction on rotating machinery, a great deal of attention has been paid to characterizing the dynamic behavior of angular contact ball bearings. As such components lay in the load transmission path between rotating and stationary parts, the correct definition of the rolling element and raceways interactions is crucial for properly characterizing the dynamic behavior of the rotating system. Considering that most vibration problems of rotating machinery are related to inherent variable compliances of the bearings, rotor and housing iterations [1], the characterization of the factor affecting such variable stiffness and damping of the contacts is essential.

Over the years, several authors have investigated, experimentally and theoretically, the dynamics of ball bearings. Early studies date back to the fundamentals of fatigue life prediction [2–4]. Harris and Mindel [5] presented the underlying development of classical ball bearings dynamics, including static and dynamic load distributions, accounting for all forces and moments acting on each bearing component. At the core of the model lays the dry contact formulation of Hertz [6] to deal with the element to raceway normal contact. Several authors [7–9] later expanded this classical model technique.

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Nomenclature

- Hertzian contact length а
- Α Distance between centers of curvature of raceways
- b Hertzian contact width C Contact Damping
- d Exponent of the displacement in the expression of contact force
- Pitch diameter d_m D Ball diameter Degree of freedom DOF
- E'Reduced modulus of elasticity
- **EHD** Elastohydrodynamic F EHD contact force f Osculation F_c Inertial force Film thickness h
- Approach between rigid bodies h_o Dimensionless film thickness Η
- Κ Contact stiffness
- Dimensionless Moes lubricant parameter L
- Ball mass m
- Dimensionless Moes load parameter Μ
- M_g Gyroscopic moment
- Pressure distribution on contact p
- Q Contact force
- Raceways radius of curvature r R_{χ} Curvature ratio in x-direction R_{ν} Curvature ratio in y-direction
- Radius of curvature in x-direction of solid 1 $R1_x$ Radius of curvature in x-direction of solid 2 $R2_x$ Radius of curvature in y-direction of solid 1 $R1_{\nu}$ Radius of curvature in y-direction of solid 2 $R2_{\nu}$
- Time t
- Relative velocity tangential to the contact area U_{m}
- Coordinate in flow direction х Ball equilibrium position Χ Coordinate perpendicular to x y
- Z Number of balls Absolute viscosity η Fluid density ρ ΔF Offset force δ Displacement
- δ Velocity
- θ Angular displacement α^0 Nominal contact angle
- ψ Azimuth angle
- Contact angle α
- \Re_I Distance between the geometric center of the bearing and the center of curvature of the raceways

Subscripts

- i Inner raceway
- Outer raceway 0
- Radial direction r
- Axial direction а
- x-direction χ
- y-direction y
- z-direction z j-th ball
- j
- Central С

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